



ANSYS

CONVERGENCE
CONFERENCES

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某型航空产品齿轮箱壳体拓扑优化设计



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行业与产品概述

• 行业背景

- 轻量化设计要求更加严格
- 质量、可靠性要求教高
- 研制周期正在压缩
- 设计需求多次迭代

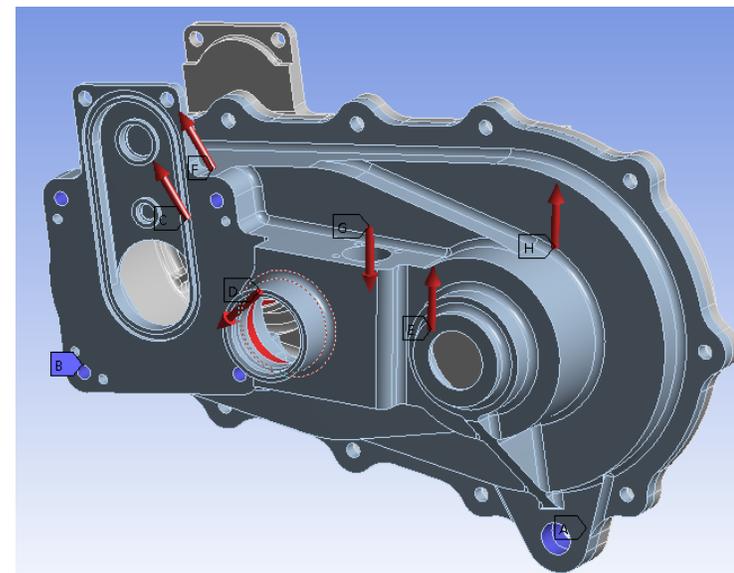
• 产品特点

- 机载设备
- 设计安全裕度较小
- 载荷环境复杂



仿真需求分析

- **设计中的关键问题**
 - 以轻量化设计为目标
 - 满足强度条件
 - 壳体支撑刚度需要控制
- **仿真需求分析**
 - 在满足产品强度、刚度需求的前提下，设计最轻量化的壳体结构

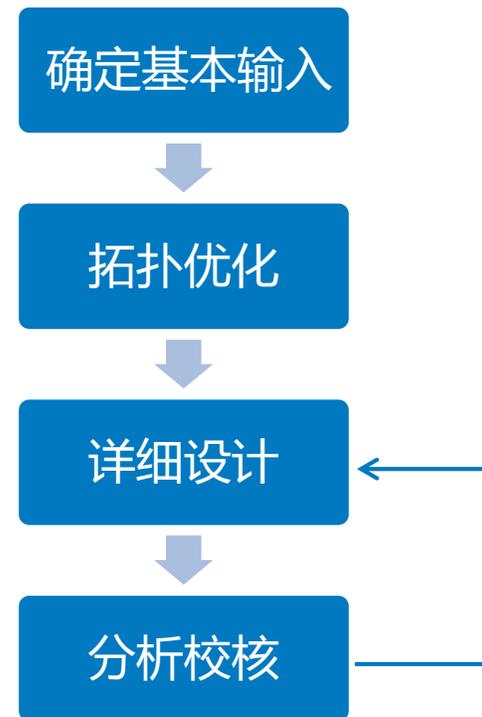


仿真设计过程简述



- **静力分析**
 - 确定边界和输入
- **拓扑优化**
 - 最小化柔顺性
 - 最小化质量：含位移约束，修改节点坐标系
- **结果验证**
 - 验证优化结果

结构优化设计思路



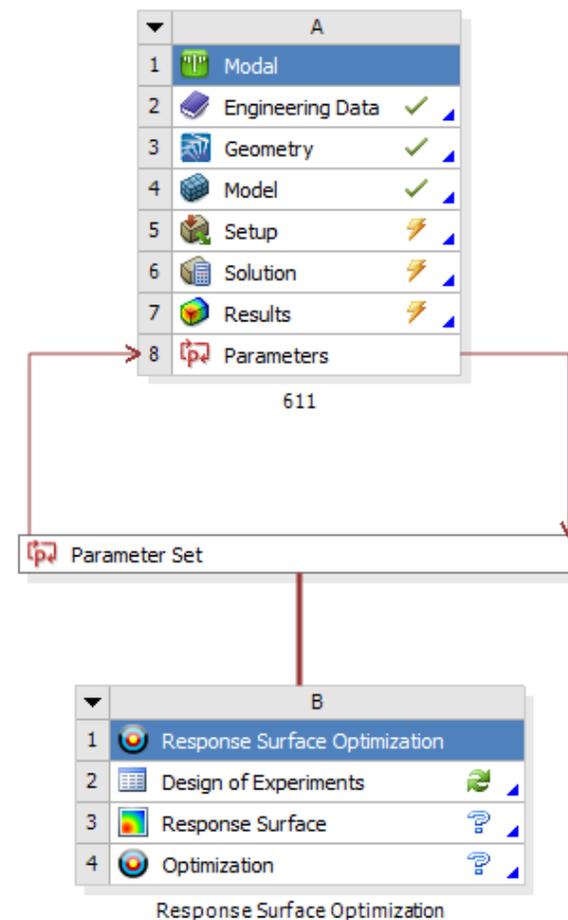
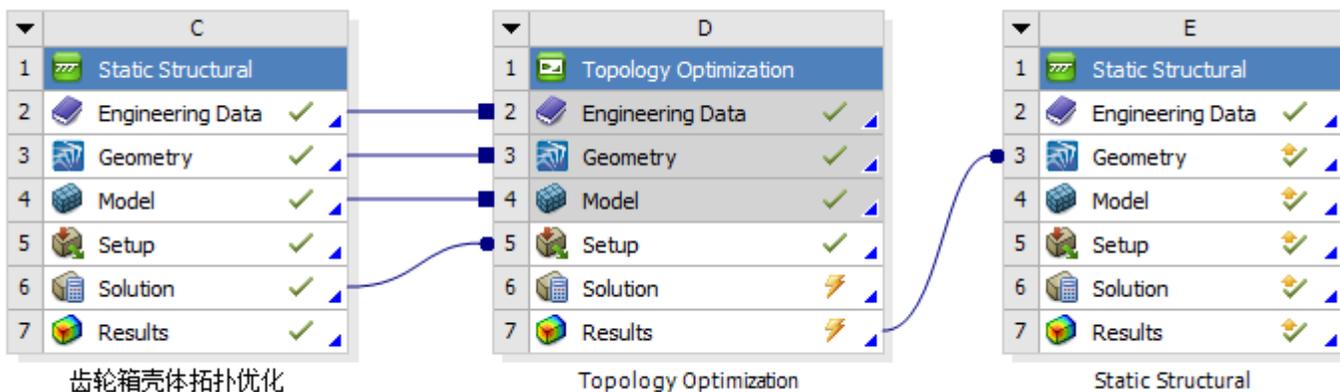
ANSYS的优化方法

- **Design Exploration**

- Direct Optimization
- Response Surface

- **拓扑优化**

- 最小化柔顺性
- 最小化质量/体积

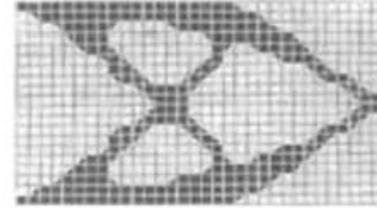


拓扑优化的一般方法

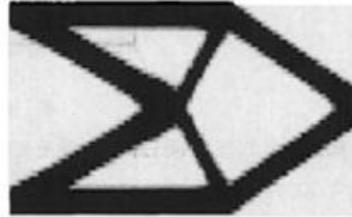
- 均匀化方法
- 变厚度法
- 变密度法
- ICM法
- ESO法
- 水平集法



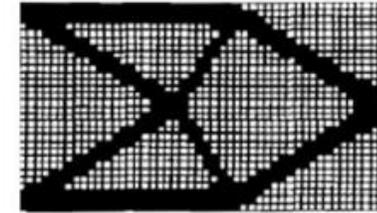
均匀化方法(50步)



变厚度法(6步)



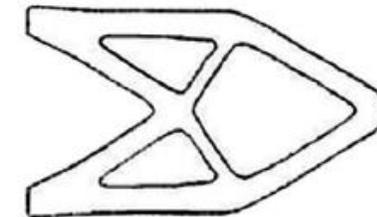
变密度法(12步)



ICM法(32步)



BESO法(33步)



水平集法(30步)

例子1 齿轮箱壳体

- **模型或数据**

- 齿轮箱壳体

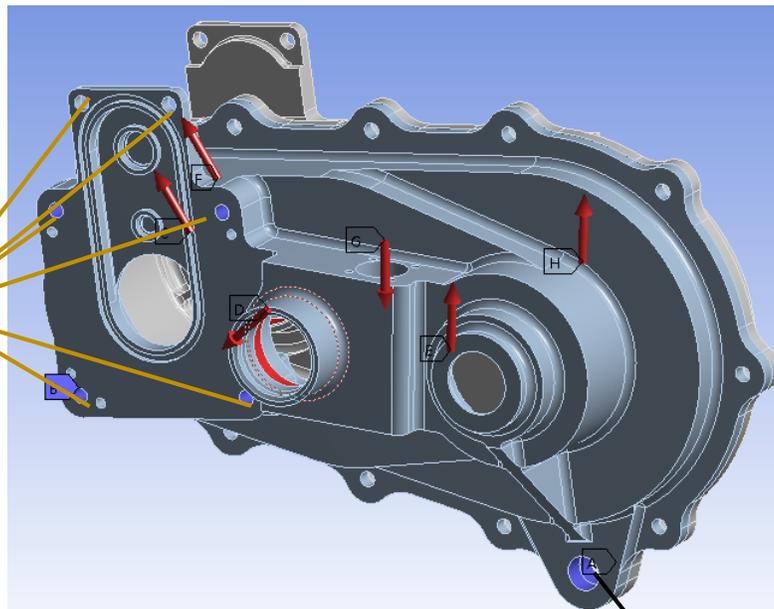
- **参数与条件**

- 边界条件：约束安装脚
- 将连接处简化为弹性支撑
- 齿轮力施加到轴承孔上
- 材料：7A09

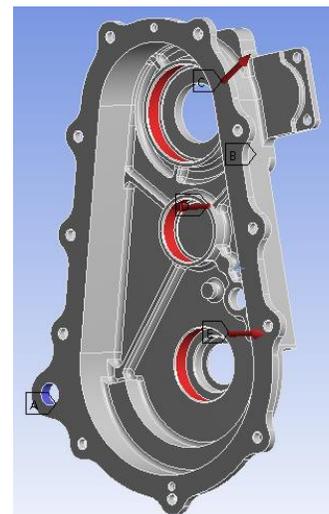
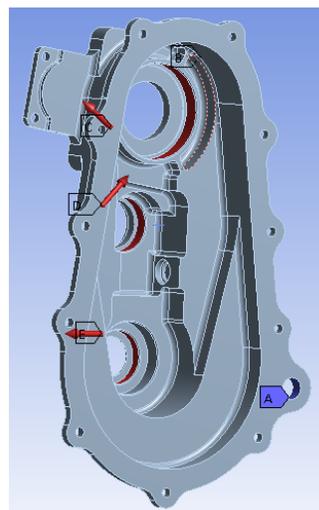
- **软件**

- ANSYS Workbench 18.1

弹性支撑

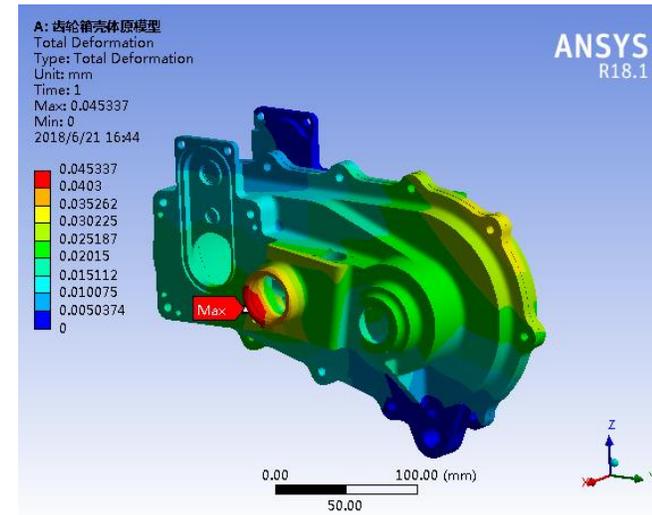
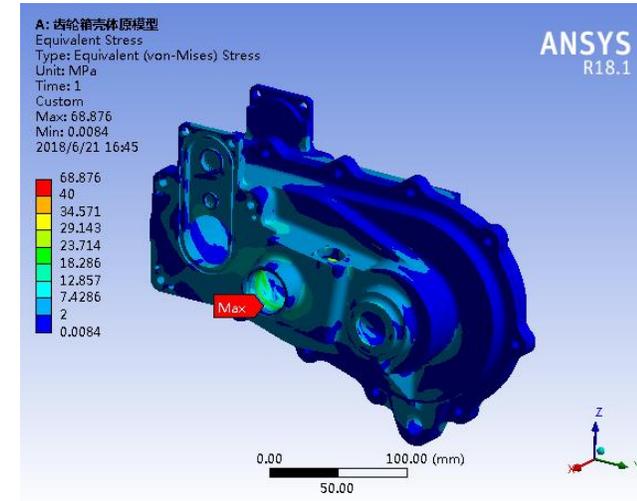
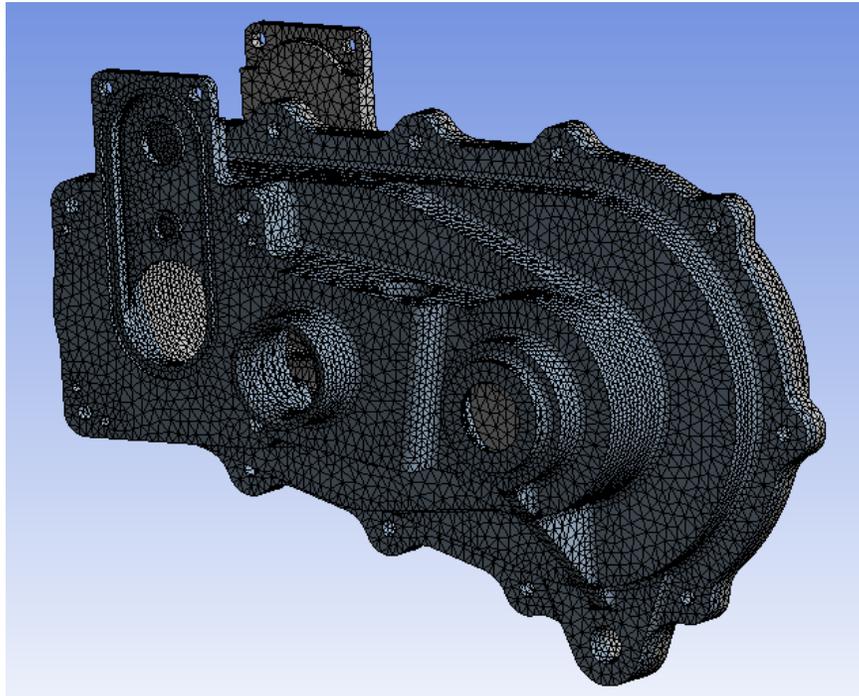


安装脚



内部齿轮力以Force施加
(注：Workbench拓扑优化不支持bearing load)

静力分析



壳体变形过大，会影响齿轮传动。因此，在减重的同时，需要控制壳体刚度。

应力

位移

拓扑优化



- **最小柔顺性**
- 优化目标：柔度最小
- 约束：体积/质量百分比

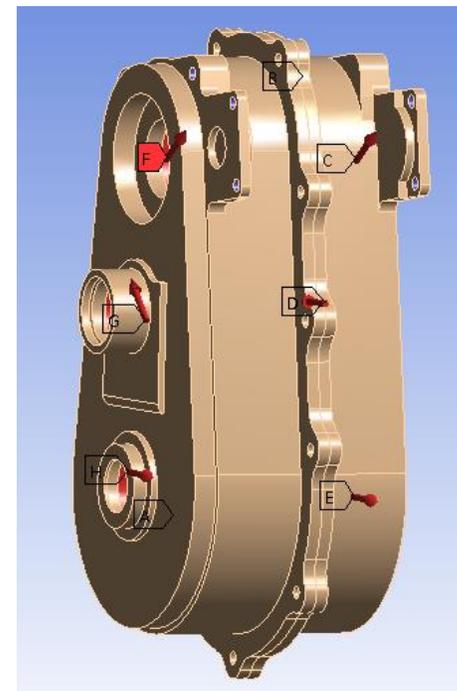
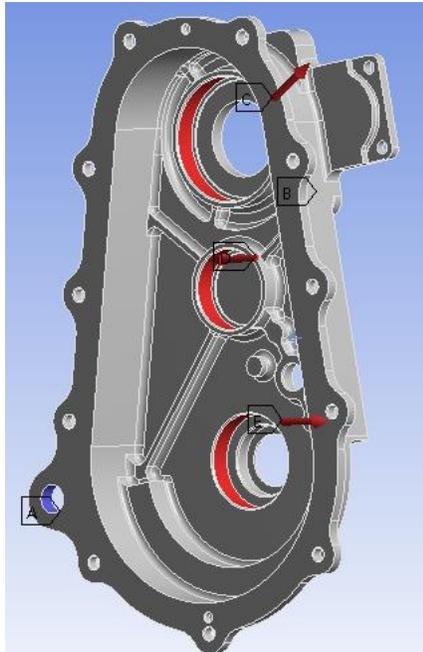
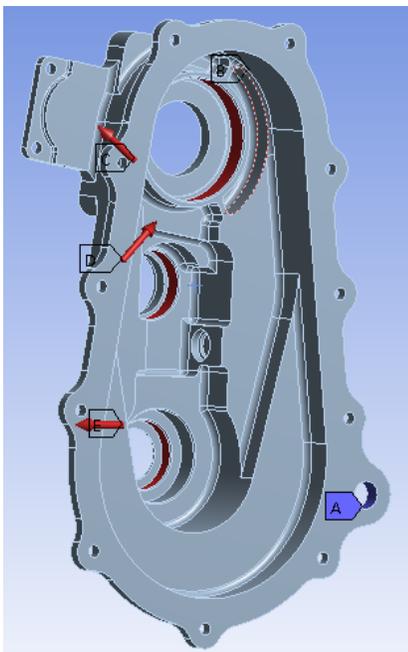
Enabled	Response Type	Goal	Formulation	Environment Name	Weight	Multiple Sets	Start Step	End Step	Step	Start Mode	End Mode	Mode
<input checked="" type="checkbox"/>	Compliance	Minimize	Program Controlled	Static Structural	1	Enabled	1	1	1	N/A	N/A	N/A

- **最小质量/体积**

- 优化目标：体积/质量最小
- 约束：应力约束、位移约束



拓扑优化



齿轮箱壳体

增材后

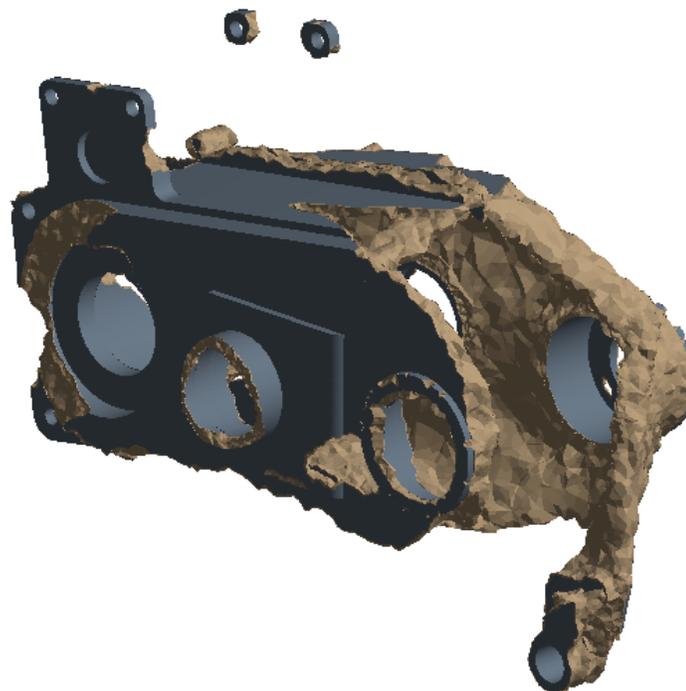
为了充分探究结构可能的拓扑形式，先对壳体进行了增材处理。

后续产品设计，应先进行整体拓扑结构设计，再进行细节设计。

拓扑优化

- 最小柔顺性的优化

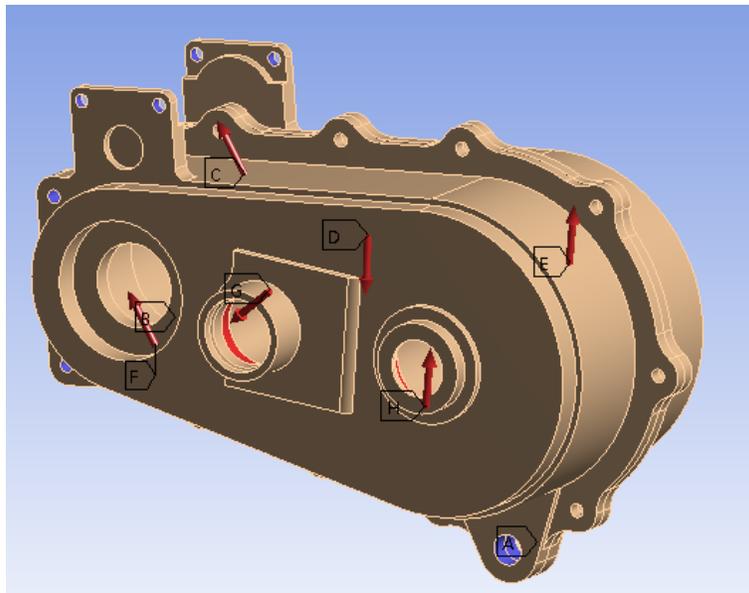
优化目标：柔度最小
约束：体积/质量百分比



约束：20%质量

拓扑优化

- 最小化质量



目标：质量最小
约束：应力

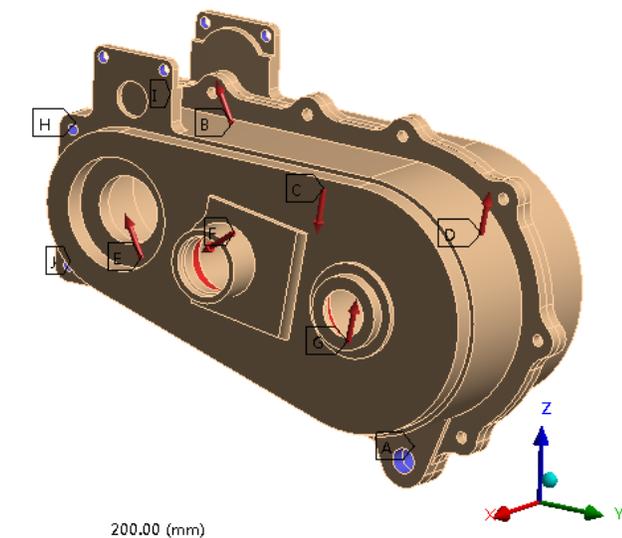


只约束应力，优化结果出现孤岛，分布在受力的轴承孔周围。

没有对位移进行限制，则应力约束也没有意义。

拓扑优化

目标：质量最小
约束1：应力
约束2：孔边位移

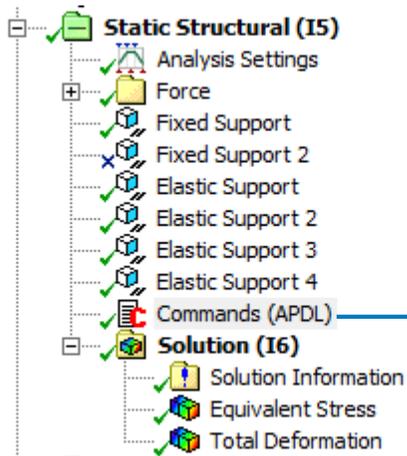


Workbench的位移约束不能选择方向和坐标系，因此当受力方向跟系统自身坐标系不一致时，位移约束就无法起作用。

Details of "Displacement Constraint 2"	
Scope	
Scoping Method	Geometry Selection
Geometry	1 Node
Definition	
Type	Response Constraint
Response	Displacement
Coordinate System	Nodal Coordinate System
X Component (Max)	Free
<input type="checkbox"/> Y Component (Max)	4.e-002 mm
Z Component (Max)	Free
Environment Selection	All Static Structural
Suppressed	No

拓扑优化

解决方式：添加命令语言。
通过添加命令，改变节点坐标系的方向，再进行位移约束。



Details of "Displacement Constraint 2"	
Scope	
Scoping Method	Geometry Selection
Geometry	1 Node
Definition	
Type	Response Constraint
Response	Displacement
Coordinate System	Nodal Coordinate System
X Component (Max)	Free
<input type="checkbox"/> Y Component (Max)	4.e-002 mm
Z Component (Max)	Free
Environment Selection	All Static Structural
Suppressed	No

NMODIF, 15735, , , , -180,

节点号

绕轴旋转角度

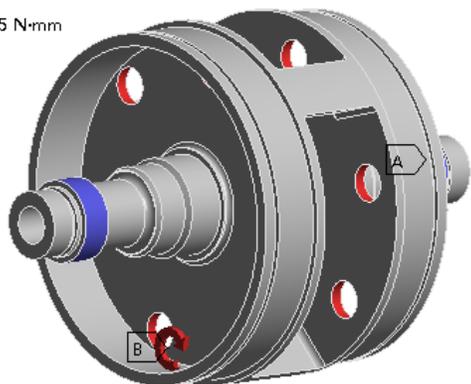


参考优化结果在中间齿轮周围布置了加强筋，减重的同时保证了壳体刚度。

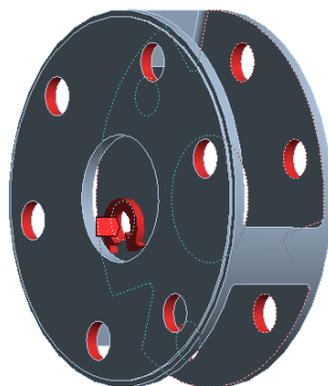
例子2 行星架方案设计

如下图所示的行星架，两端通过轴承固定，中间连接行星轴，轴上承受扭矩。

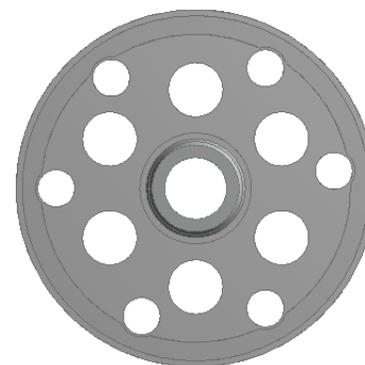
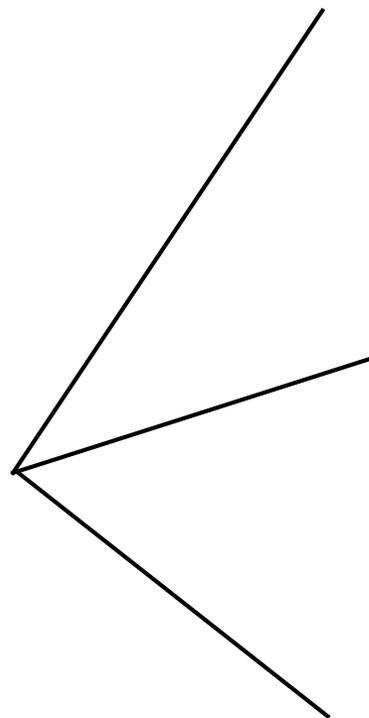
A: Static Structural
Static Structural
Time: 1. s
2018/7/2 19:12
A Fixed Support
B Moment: 2.e+005 N·mm



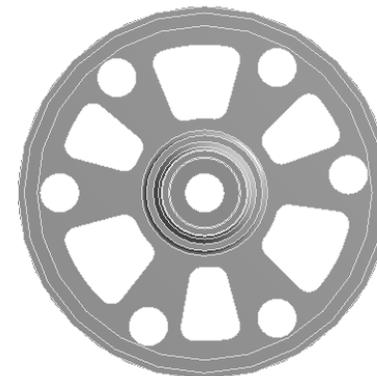
0.00 100.00 (mm)
50.00



优化区域



方案一



方案二

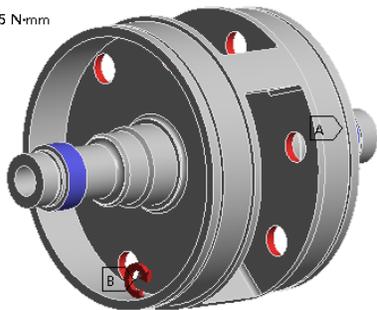


拓扑优化

静力分析

A: Static Structural
Static Structural
Time: 1. s
2018/7/2 19:12

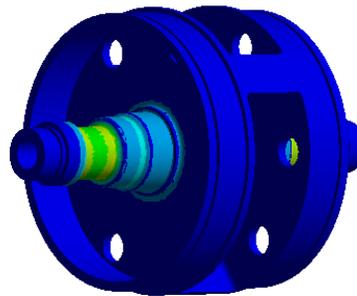
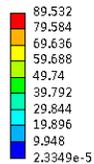
- Fixed Support
- Moment: 2.e+005 N-mm



0.00 100.00 (mm)
50.00



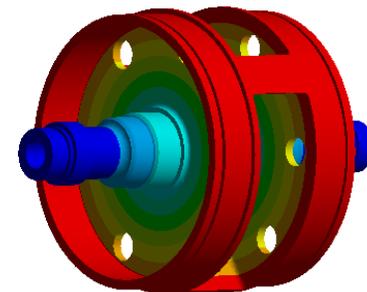
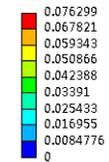
A: Static Structural
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1
Custom
Max: 89.532
Min: 2.3349e-5
2018/7/2 19:35



0.00 100.00 (mm)
50.00



A: Static Structural
Total Deformation
Type: Total Deformation
Unit: mm
Time: 1
Custom
Max: 0.076299
Min: 0
2018/7/2 19:37



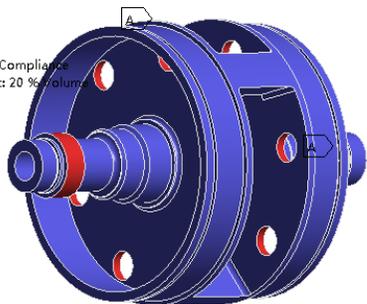
0.00 100.00 (mm)
50.00



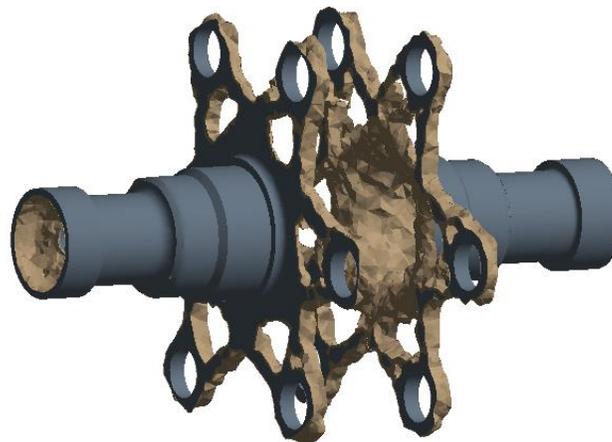
拓扑优化

B: Topology Optimization
Topology Optimization
Iteration Number: N/A
2018/7/2 19:39

- Design Region
- Exclusion Region
- Objective: Minimize Compliance
- Response Constraint: 20 % Volume

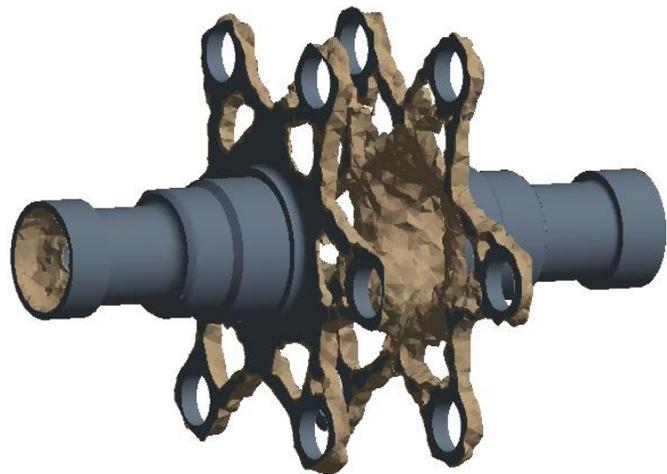


0.00 100.00 (mm)
50.00



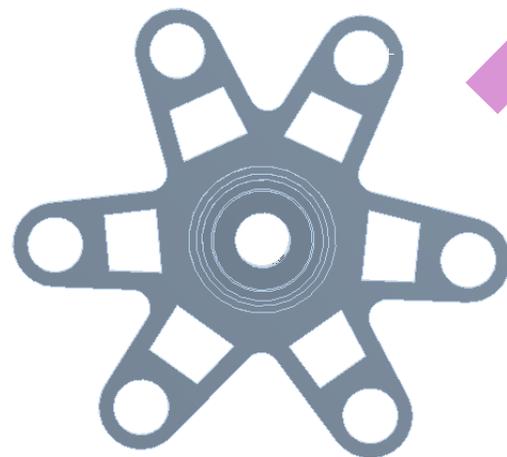
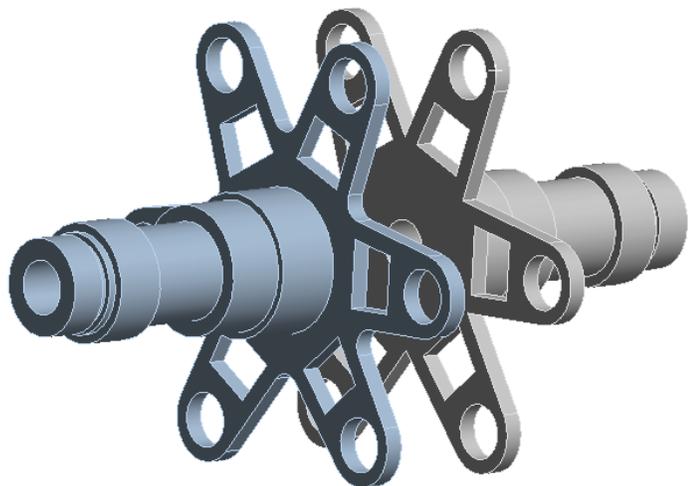
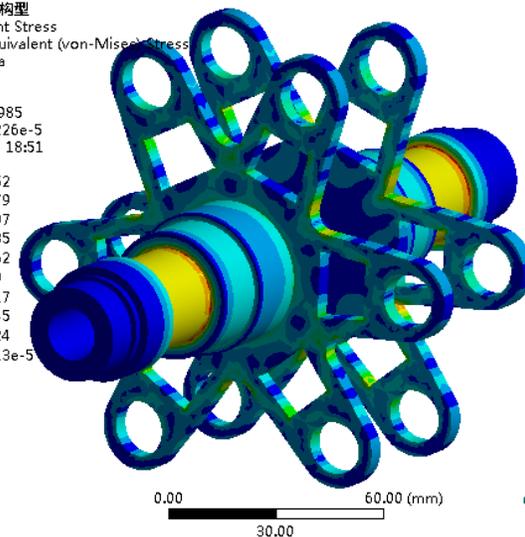
以柔顺性最小为目标，20%的体积约束

优化后处理



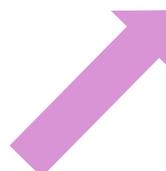
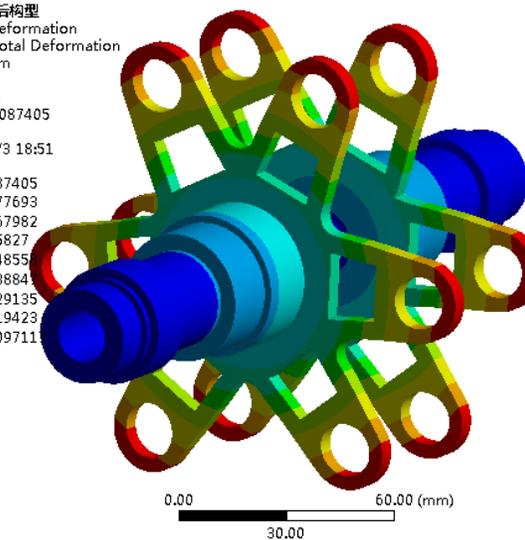
K: 拓扑后构型
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1
Custom
Max: 67.985
Min: 2.1226e-5
2018/7/3 18:51

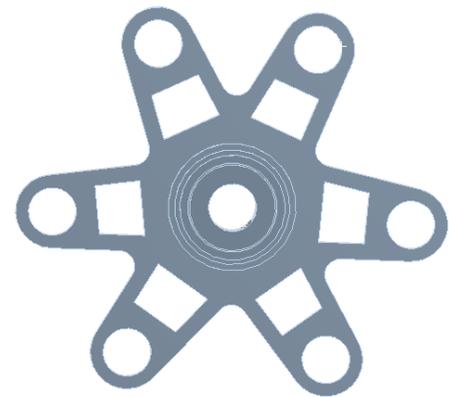
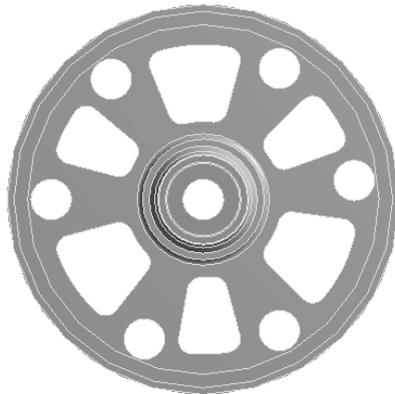
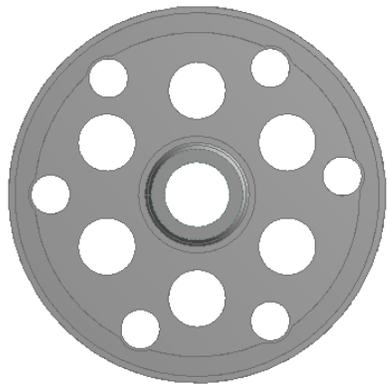
67.252
59.779
52.307
44.835
37.362
29.89
22.417
14.945
7.4724
2.6113e-5



K: 拓扑后构型
Total Deformation
Type: Total Deformation
Unit: mm
Time: 1
Custom
Max: 0.087405
Min: 0
2018/7/3 18:51

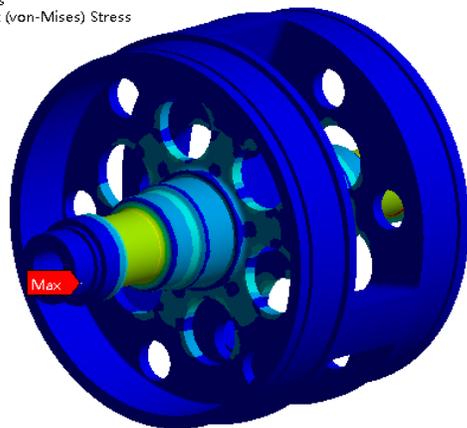
0.087405
0.077693
0.067982
0.05827
0.04855
0.03884
0.029135
0.019423
0.009711
0





B: 11
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1
Custom
Max: 87.884
Min: 4.2269e-5
2018/7/3 18:58

87.884
78.119
68.354
58.589
48.825
39.06
29.295
19.53
9.7649
4.2269e-5

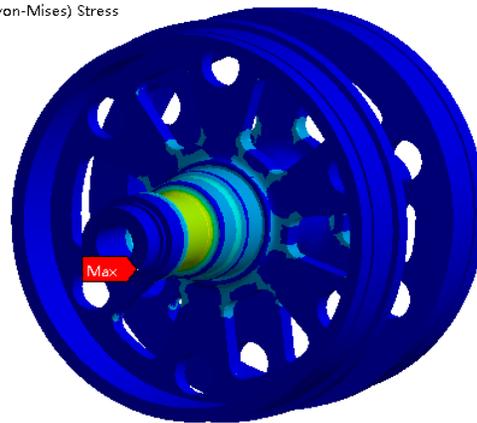


0.00 80.00 (mm)
40.00



C: 33
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1
Custom
Max: 88.928
Min: 4.6024e-5
2018/7/3 19:00

88.928
79.047
69.166
59.285
49.404
39.524
29.643
19.762
9.8809
4.6024e-5

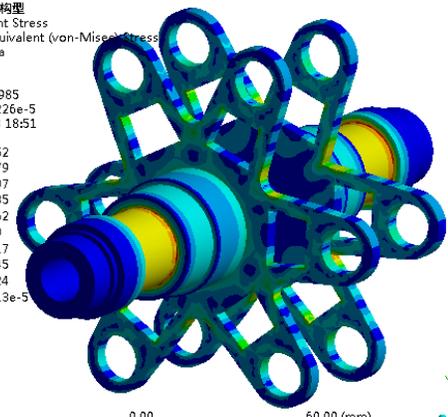


0.00 80.00 (mm)
40.00



K: 拓扑后模型
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1
Custom
Max: 67.985
Min: 2.1226e-5
2018/7/3 18:51

67.252
59.779
52.307
44.835
37.362
29.89
22.417
14.945
7.4724
2.6113e-5



0.00 60.00 (mm)
30.00



质量 3.25kg
最大应力 89MPa
最大变形 82μm

质量 3.16kg
最大应力 89MPa
最大变形 79μm

质量 1.37kg
最大应力 67MPa
最大变形 87μm

小结



- **可通过在workbench中添加命令的方式，来合理设置计算条件**
- **ANSYS拓扑优化会是结构优化设计的一个有力工具，在产品流程早期使用效果更佳。**

THANK YOU

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感谢聆听！

